

POLITECNICO
MILANO 1863

The role of CFD for the design of hydrogen propulsion systems: developments and applications @PoliMi

G. Montenegro

IDROGENO E TECNOLOGIE PER LA GENERAZIONE ENERGETICA E LA PROPULSIONE NEI TRASPORTI GREEN

Genova, 25 Gennaio 2024

1. ICEGroup
2. 1D and 3D tools developed @PoliMi
3. Applications of CFD in the field of H2 for propulsion systems
 - System modeling (engine/pumping systems)
 - Injection
 - Combustion
 - Aftertreatment
 - Fuel cells

ICE Group



Industrial collaborations



Funded project (national)

- AMAZING (PoliMI - Stems)
- Flex-Gen (STEMS - PoliMi-PoliTo)
- DHyCE-HD (PoliBa - PoliMi)

- Fundamental and applied research on thermal and fluid-dynamics modeling of energy conversion components

- IC engines
- Fluid Machines
- Electrification
- Fuel-cells

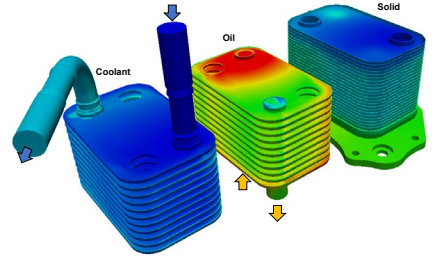
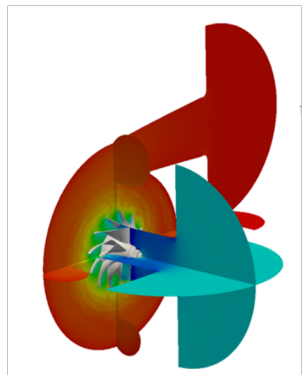
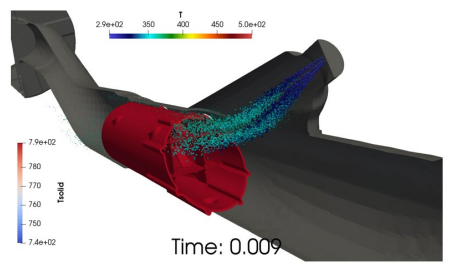
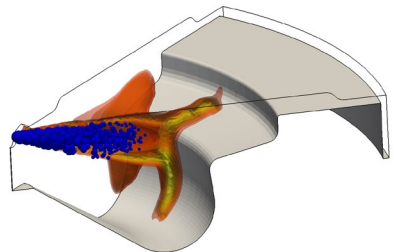
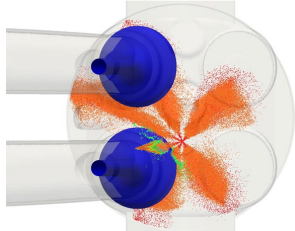
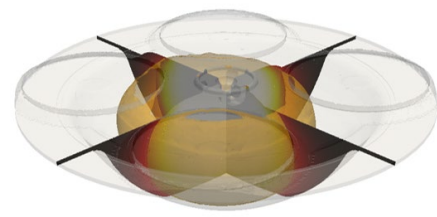
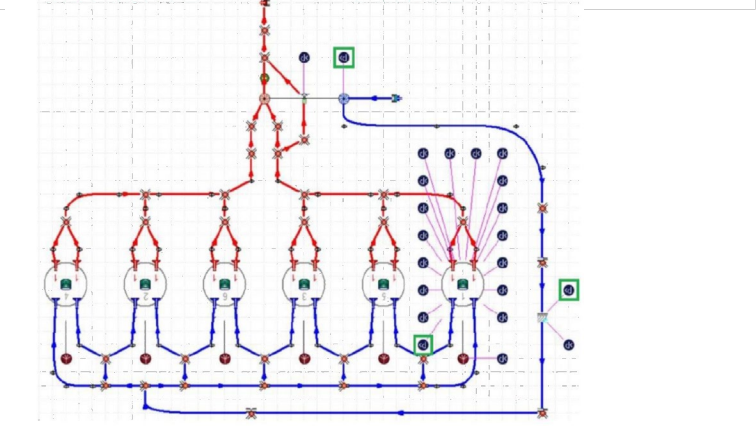
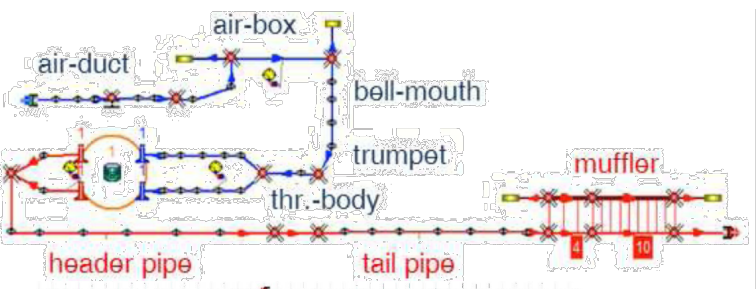
- 5 professors, 2 assistant professors, 6 PhD students;

- 2-3 visiting Phd students from Europe;

- More than 250 publications in peer-review journals and international conferences;

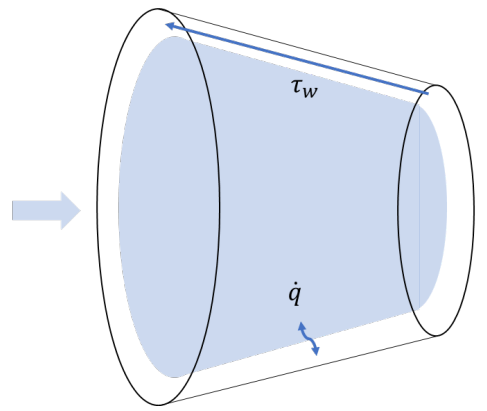
- Co-founders of the Innovative Spin-off company Sursum-Mi

Numerical modeling of IC engines, fluid machines and power systems



- **One-dimensional, unsteady, compressible, reacting flow with area variations, friction and heat transfer with the walls**

$$\begin{cases} \frac{\partial(\rho F)}{\partial t} + \frac{\partial(\rho u F)}{\partial x} = 0 \\ \frac{\partial(\rho u F)}{\partial t} + \frac{\partial(\rho u^2 + p)F}{\partial x} - p \frac{dF}{dx} + \rho F G = 0 \\ \frac{\partial(\rho e_0 F)}{\partial t} + \frac{\partial(\rho h_0 u F)}{\partial x} - (\dot{q}\rho + \Delta H_{react})F = 0 \end{cases}$$

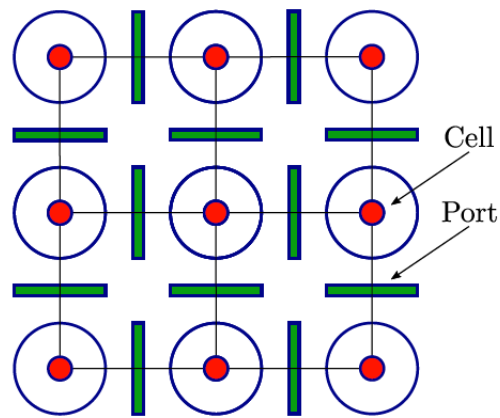


$$\begin{aligned} &\rho + \frac{\partial \rho}{\partial x} dx \\ &u + \frac{\partial u}{\partial x} dx \\ &p + \frac{\partial p}{\partial x} dx \\ &F + \frac{\partial F}{\partial x} dx \end{aligned}$$

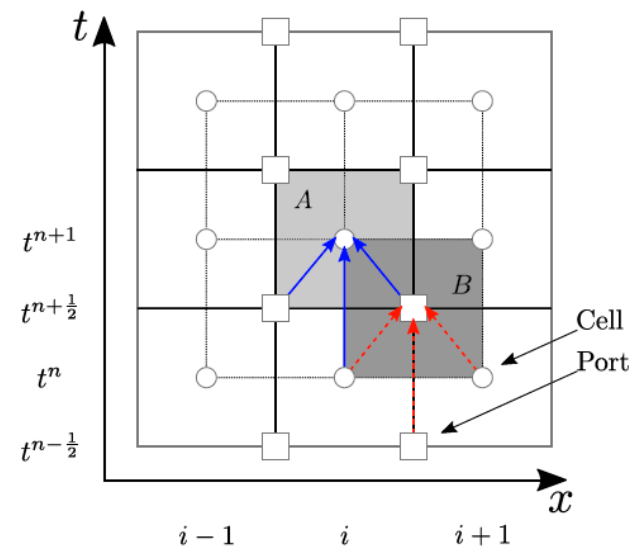
$$W(x, t) = \begin{bmatrix} \rho F \\ \rho u F \\ \rho e_0 F \\ \rho Y F \end{bmatrix}; F(W) = \begin{bmatrix} \rho u F \\ pF + \rho u^2 F \\ \rho u h_0 F \\ \rho u Y F \end{bmatrix};$$

$$B(W) = \begin{bmatrix} 0 \\ -p \frac{dF}{dx} \\ 0 \\ 0 \end{bmatrix}; C(W) = \begin{bmatrix} 0 \\ \rho G F \\ -(\rho \dot{q} + \Delta H_{react})F \\ \rho \dot{Y} F \end{bmatrix}; Y = \begin{bmatrix} Y_1 \\ \vdots \\ Y_{N-1} \end{bmatrix}$$

- **Finite volume method, explicit staggered space-time**

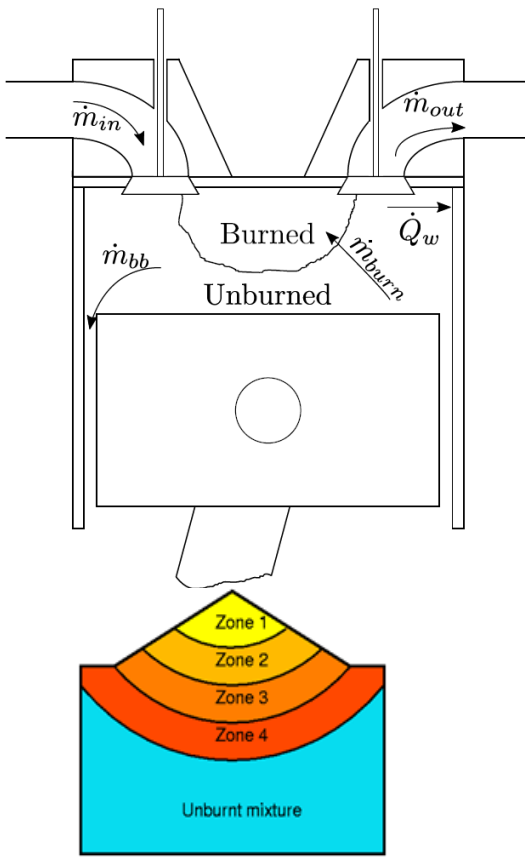


$$\begin{aligned} \rho_{cell}^{t+1} &= \rho_{cell}^t + \frac{\Delta t}{V_{cell}} \sum_p^{N_{ports}} (\rho v A)_p^{t+1/2} \\ (\rho e_0)_{cell}^{t+1} &= (\rho e_0)_{cell}^t + \frac{\Delta t}{V_{cell}} \sum_p^{N_{ports}} (\rho v h_0 A)_p^{t+1/2} + S_e \\ (\rho Y_i)_{cell}^{t+1} &= (\rho Y_i)_{cell}^t + \frac{\Delta t}{V_{cell}} \sum_p^{N_{ports}} (\rho v Y_i A)_p^{t+1/2} + S_{Y_i} \\ (\rho v A)_{port}^{t+1/2} &= (\rho v A)_{port}^{t-1/2} + \frac{\Delta t}{\Delta L_{cell}} \sum_c^{N_{cells}} [(p + \rho v^2)]_c^t \cdot A_p + S_M \end{aligned}$$



- The in-cylinder phenomena and combustion are modelled by means of **multizone** approach solving the **energy and mass balance** of each sub zone

- Zimont's **correlation** is used to compute **turbulent flame speed** $\Rightarrow S_t = u'^{3/4} L_i^{1/4} S_l^{1/2} k_u^{-1/4}$

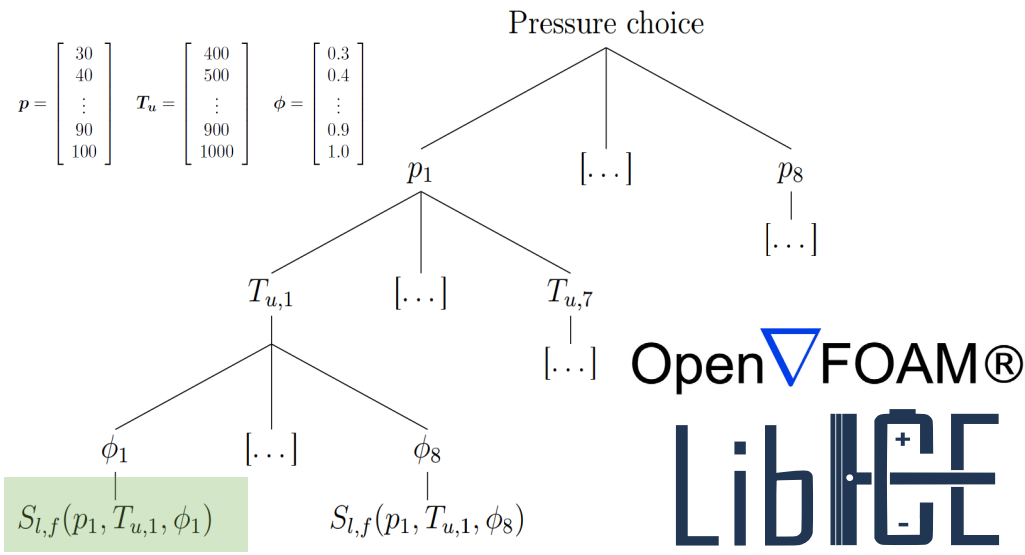


Laminar flame speed evaluation

Old method \Rightarrow Empirical correlation \Rightarrow **Strict** validity limits

New method \Rightarrow Tabulated Kinetic approach \Rightarrow **Wider** limit ranges

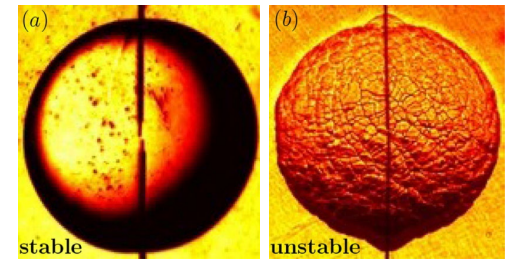
(3D matrix with around 448 values)

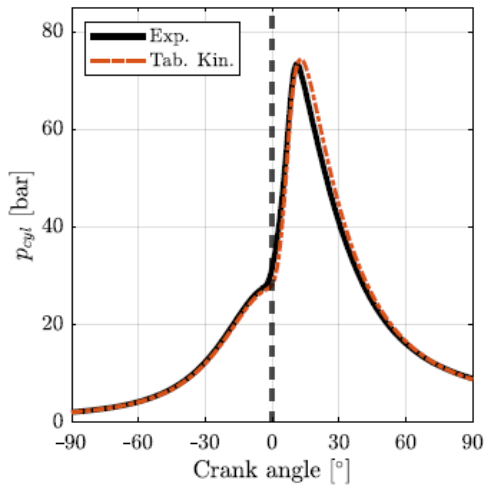


Empirical correction factor for instabilities

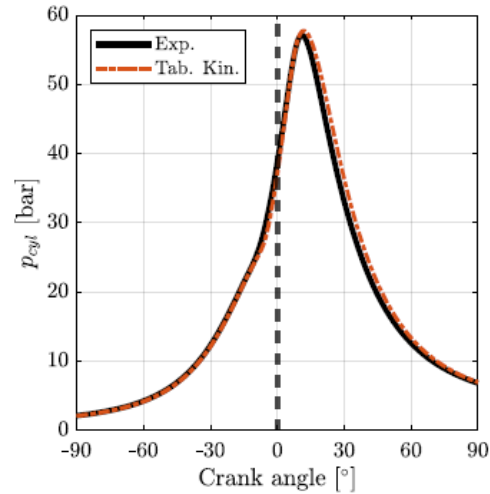
$$S_{l,c} = S_l \cdot \left(1 + \left(\left(\frac{S_{l,c}}{S_l} \right)_{ref} - 1 \right) \cdot \left(\frac{\phi}{\phi_{ref}} \right)^{\gamma_\phi} \cdot \left(\frac{T_u}{T_{u,ref}} \right)^{\gamma_{T_u}} \cdot \left(\frac{p}{p_{ref}} \right)^{\gamma_p} \right)$$

Courtesy of X. Wen, T. Zirwes, A. Scholtissek et al., Flame structure analysis and composition space modeling of thermodynamically unstable premixed hydrogen flame, <https://doi.org/10.1016/j.combustflame.2021.111808>

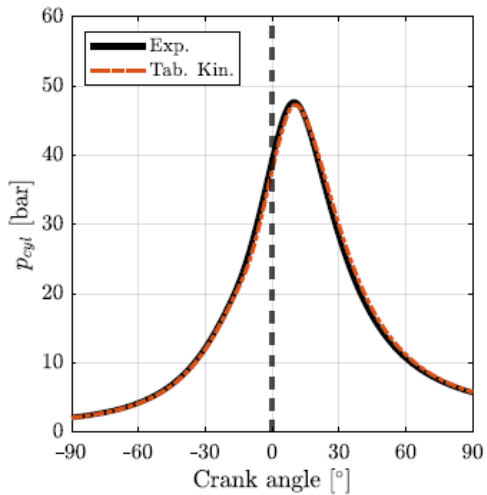




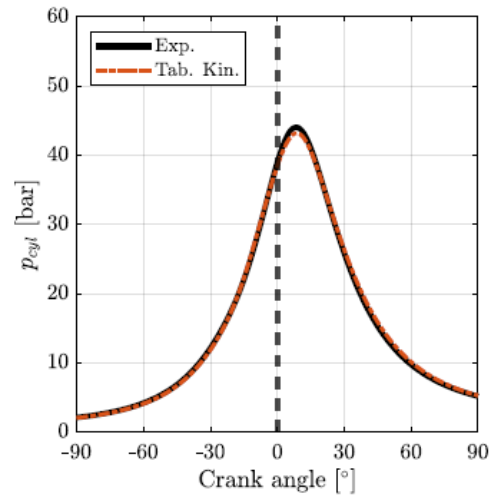
(a) $\lambda = 1.0$



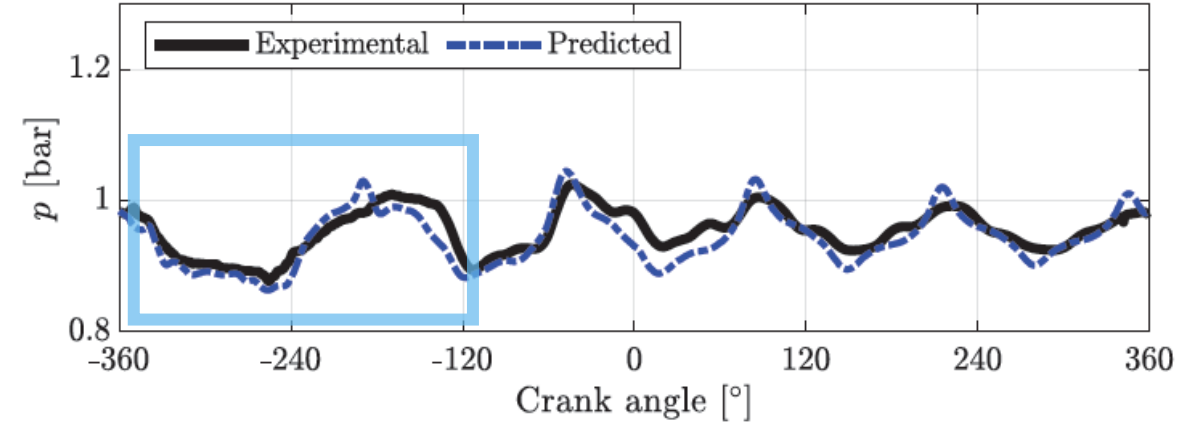
(b) $\lambda = 1.74$



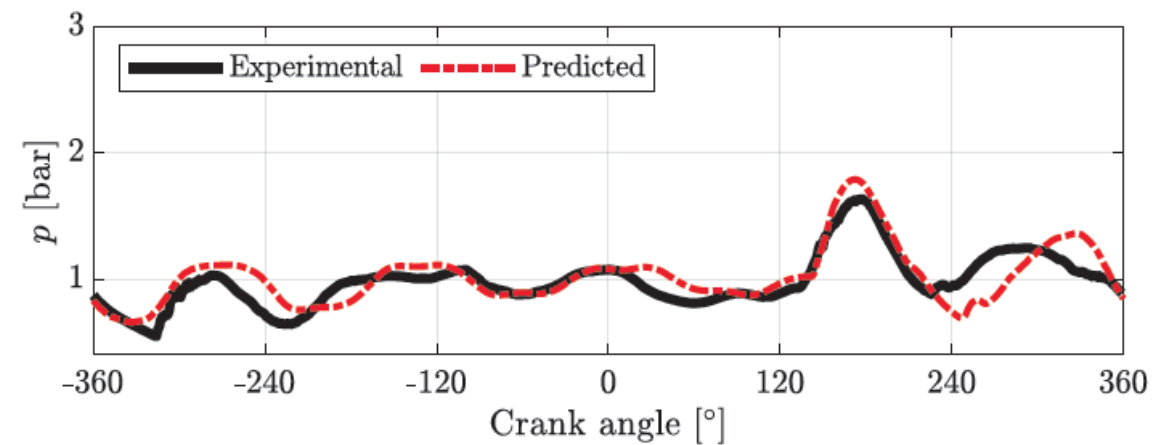
(c) $\lambda = 2.45$



(d) $\lambda = 2.96$

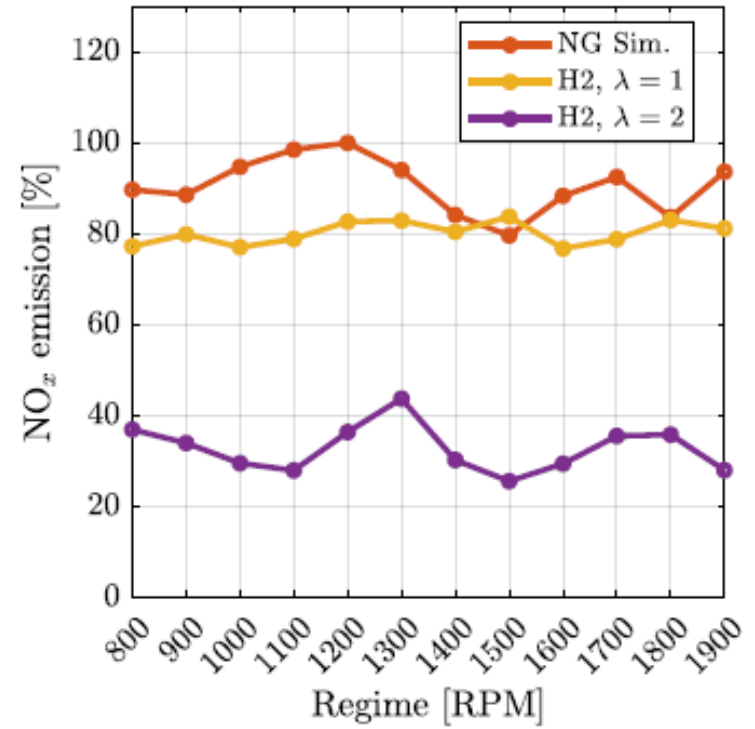
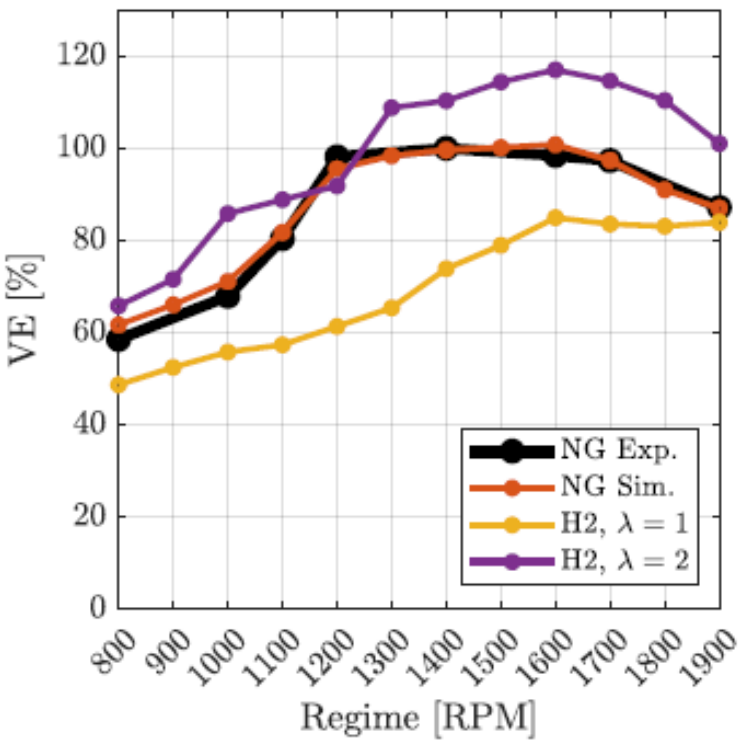
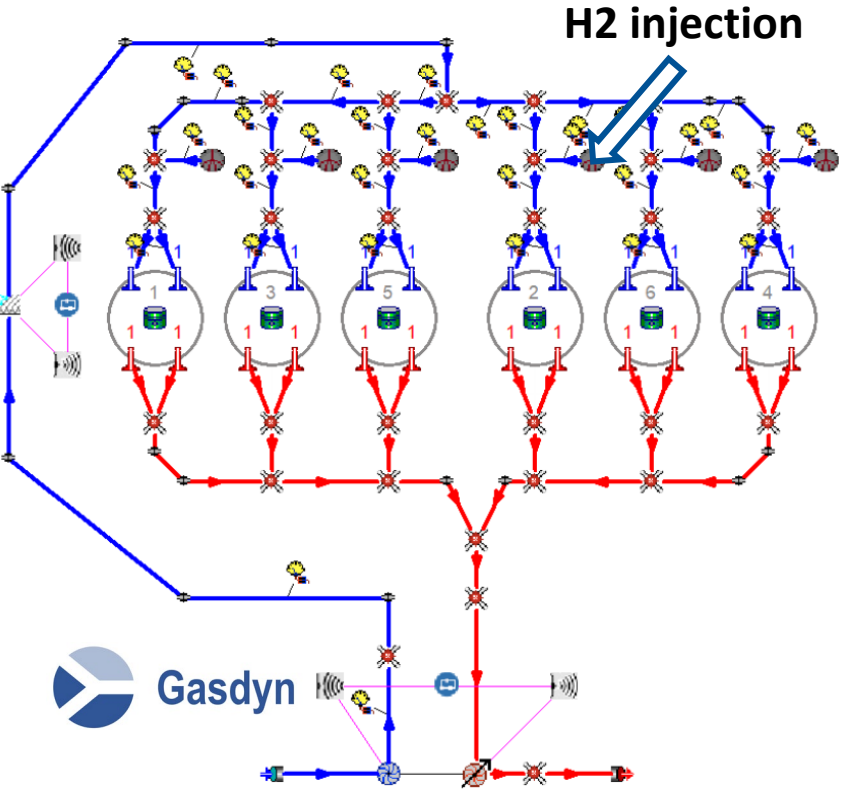


(a) Intake; $\lambda = 1.0$

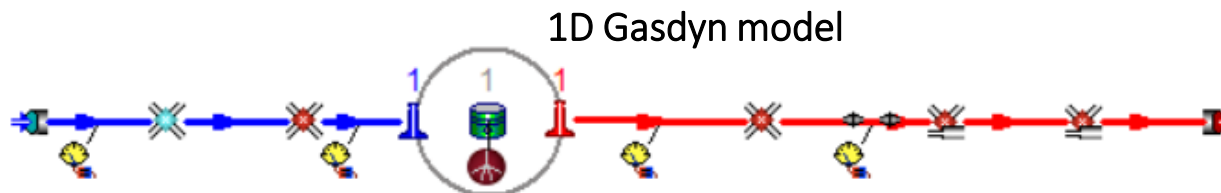
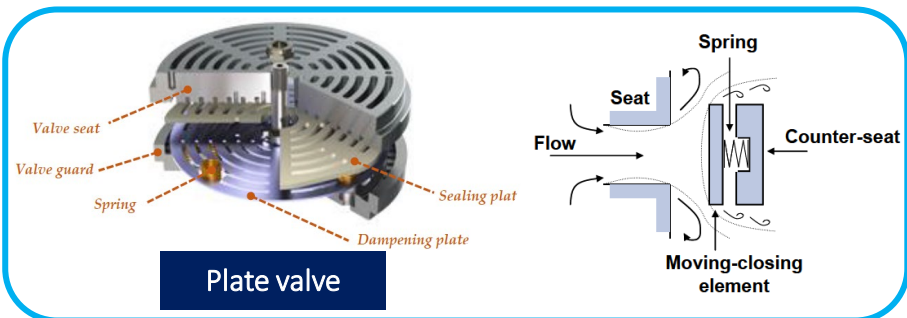


(b) Exhaust; $\lambda = 1.0$

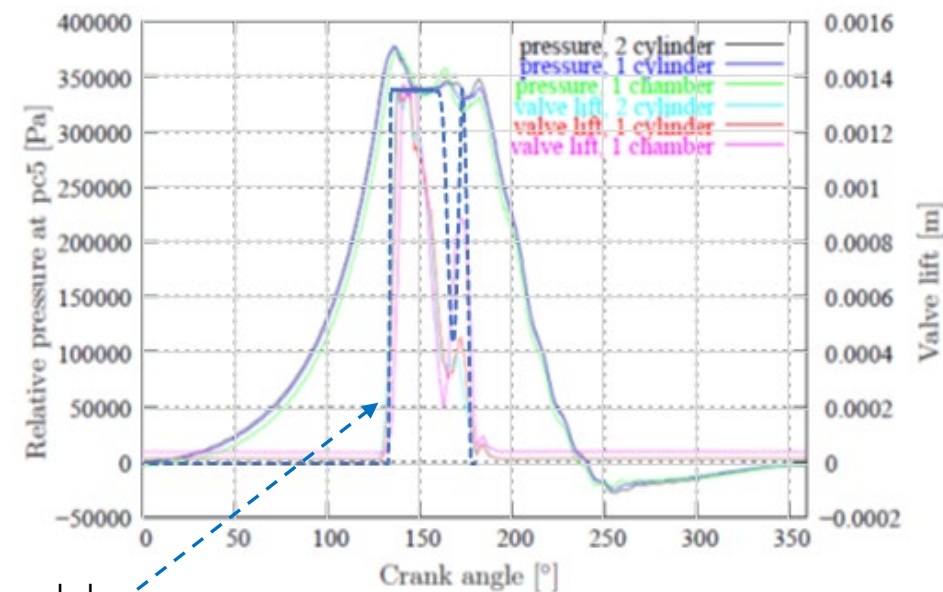
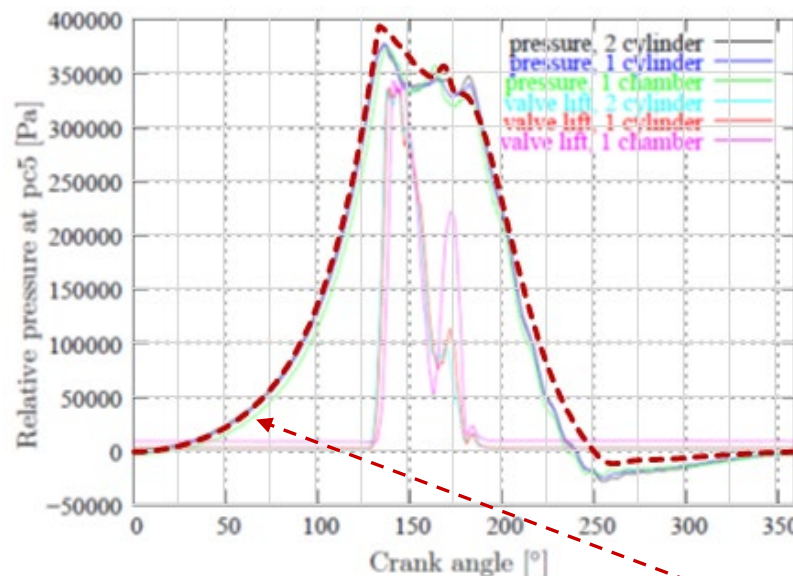
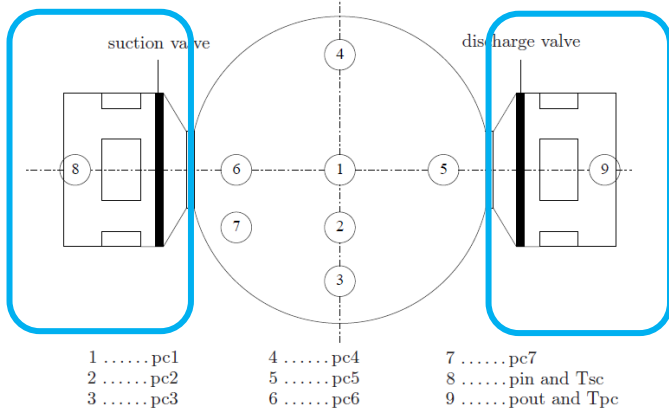
- Natural gas fuelled **heavy duty in-line 6 cylinder** featuring a **variable geometry turbine**
- $\lambda = 1$ and $\lambda = 2$ feeding conditions have been considered



Test case: *Burckhardt compressor 2K90-1A*



Comparison of relative pressure at pc5 and valve lift at different times t_a , t_b and t_c . Case: $p_{out} = 3 \text{ bar}$, $c_s = 48 \text{ N/mm}$, $x_{v,max} = 1.35 \text{ mm}$.

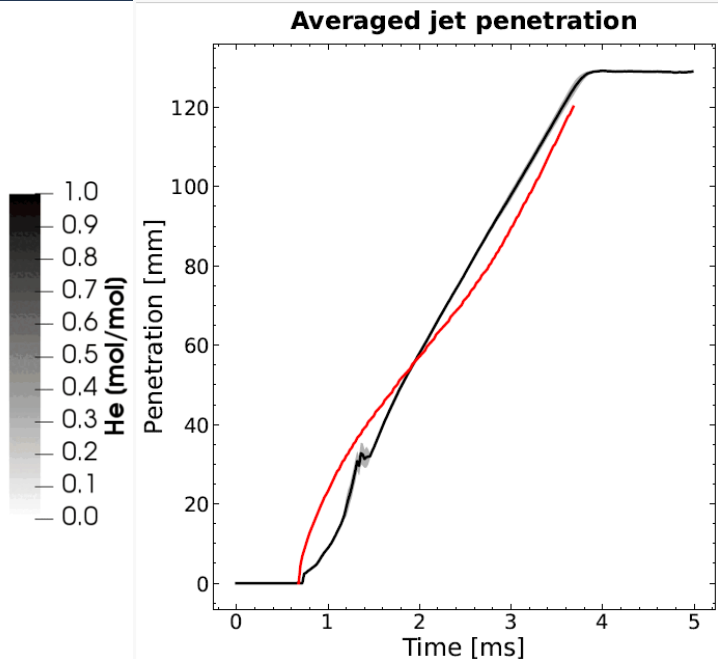
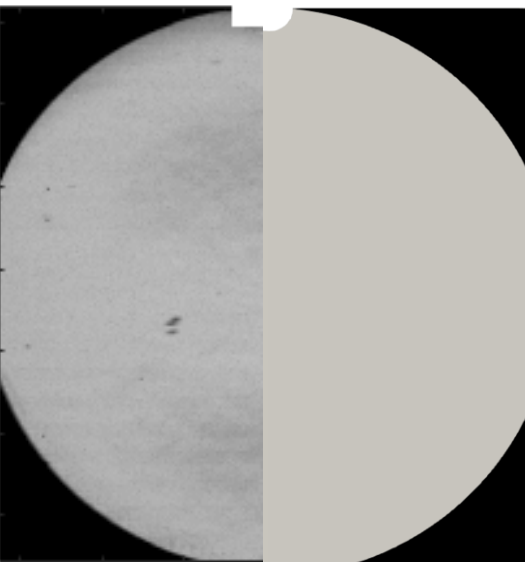


1D model predicted trace

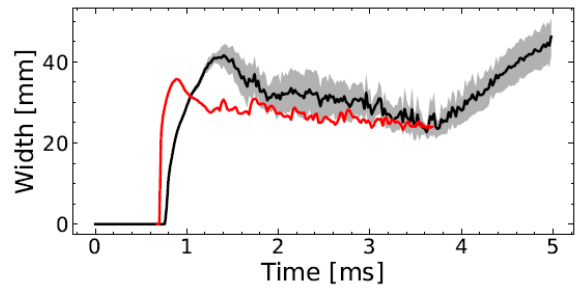
Location of the sensors inside the cylinder 1, the suction and the discharge valve retainer.

Modeling the H2 injection phase

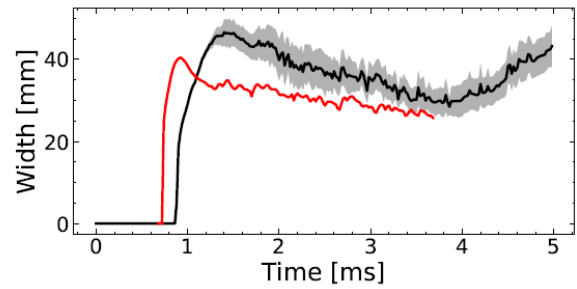
0.000 ms



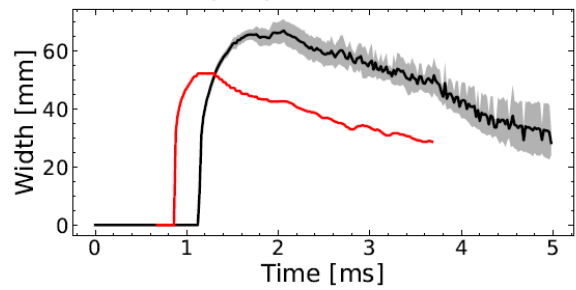
Averaged jet width at 3mm



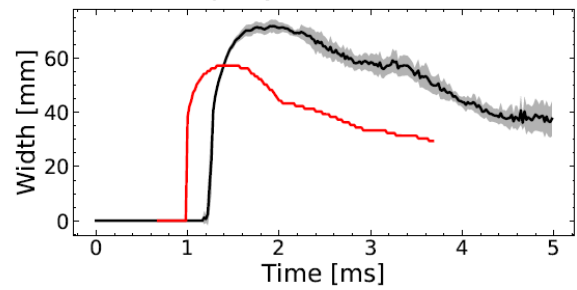
Averaged jet width at 5mm



Averaged jet width at 15mm

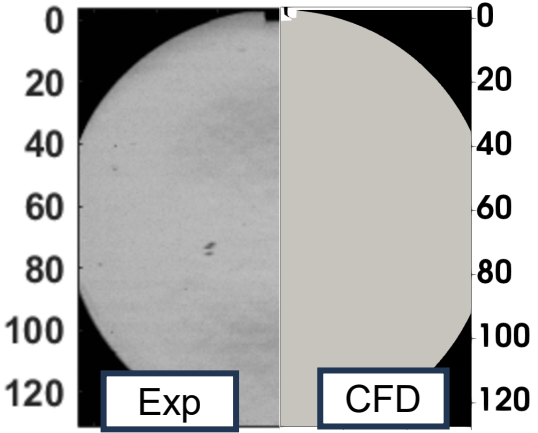


Averaged jet width at 21mm

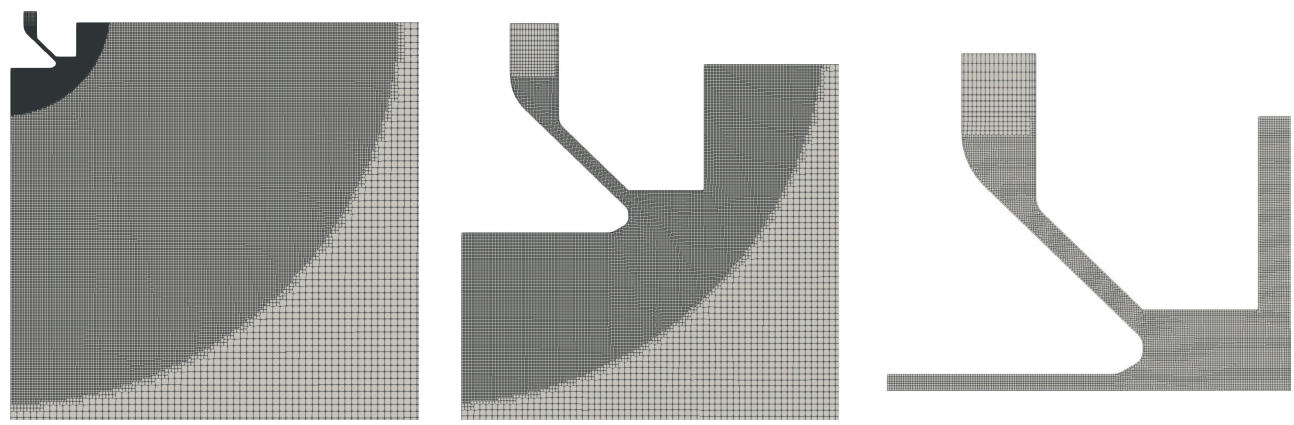
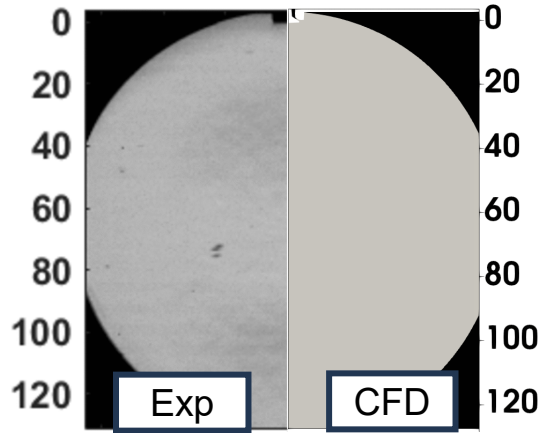


— Experimental — Calculated $x_{He}=99\%$

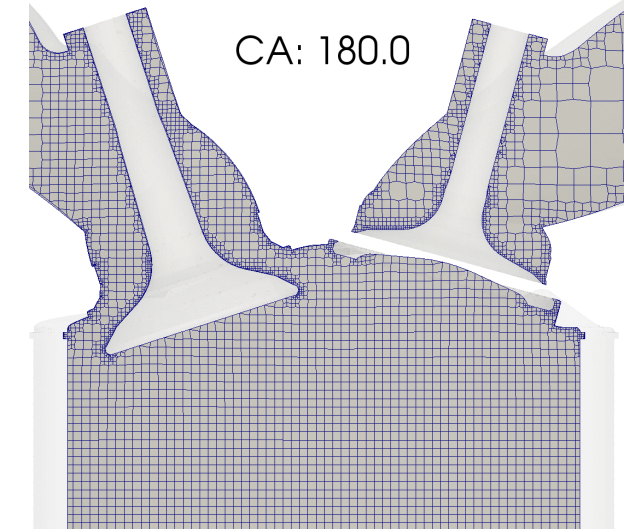
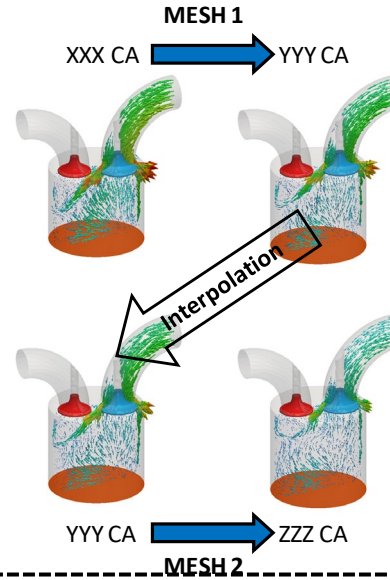
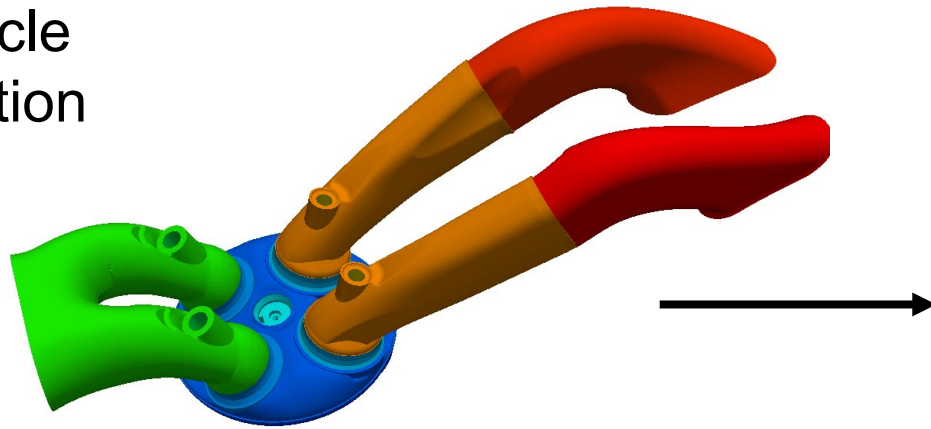
0.680 ms



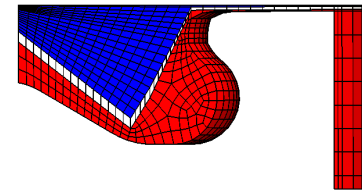
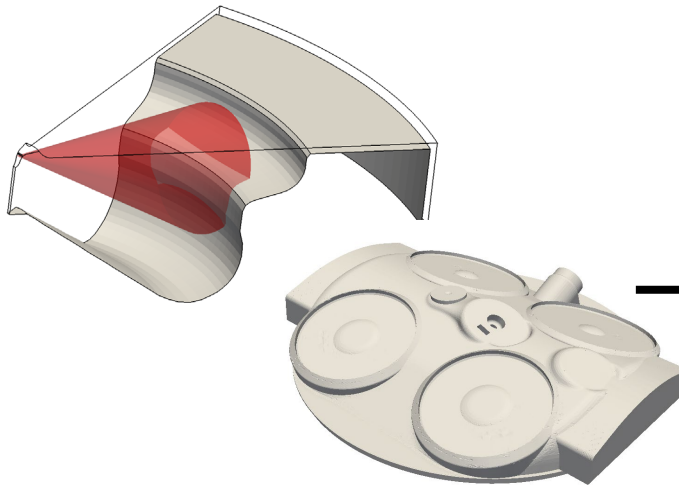
0.680 ms

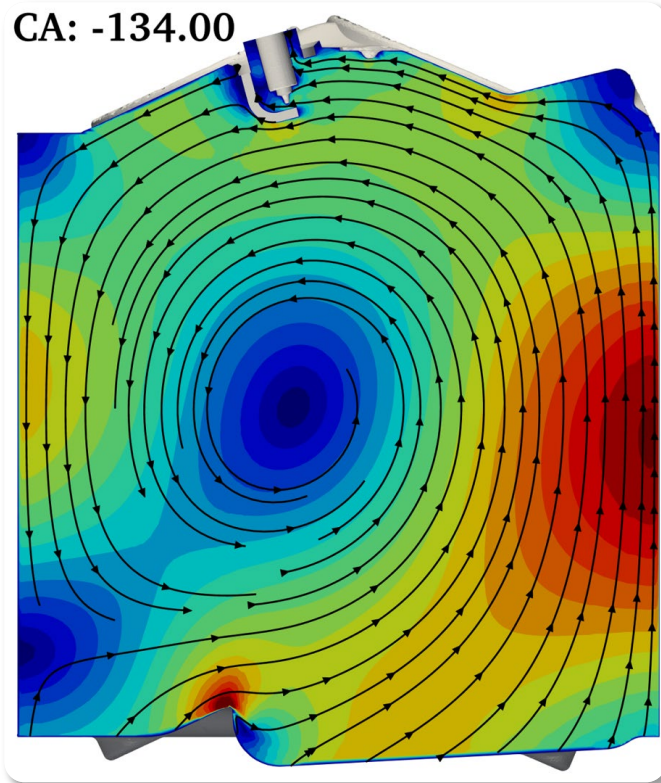


Full-cycle simulation

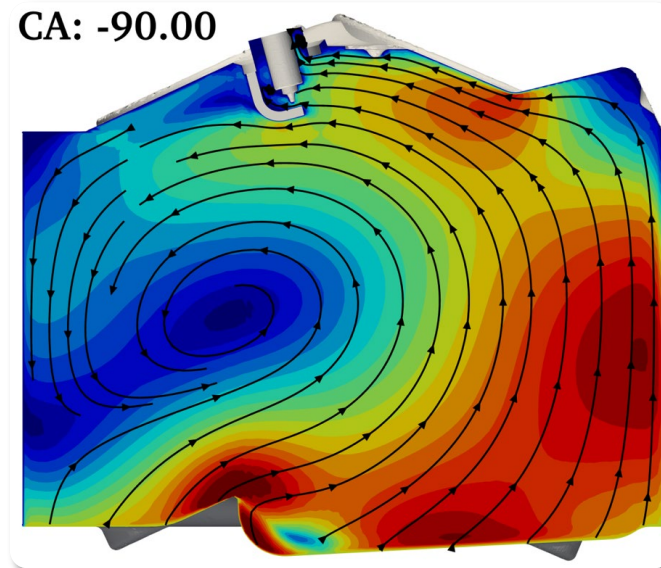


Power-cycle simulation

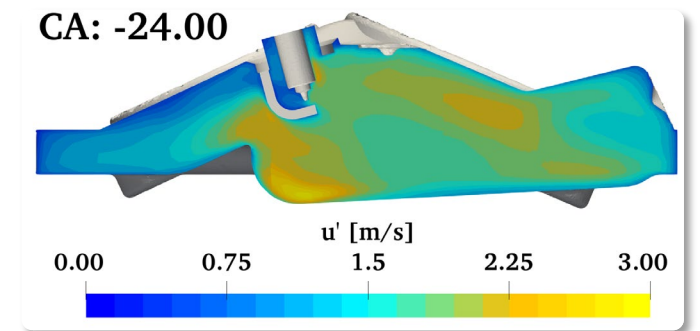
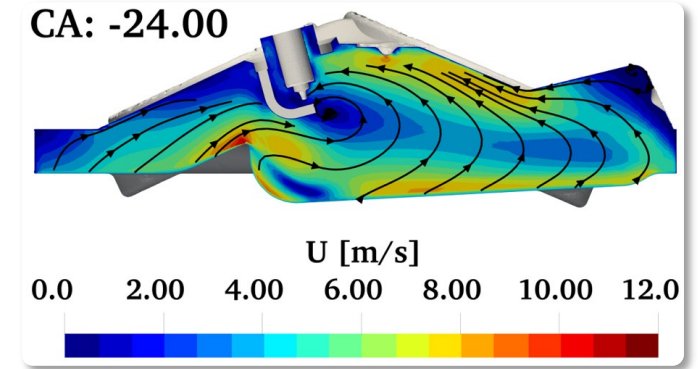




Tumble motion initialized @IVC
according to validated
methodology



Compression stage simulation



Generation of flow velocity and
turbulence fields @SA

• **Spark-ignition combustion**

- Flame area evolution (FAE) model to predict the **reaction rate**:

$$\frac{\partial \rho \tilde{b}}{\partial t} + \nabla \cdot (\rho \tilde{U} \tilde{b}) + \nabla \cdot (\mu_t \nabla \tilde{b}) = \rho_u \tilde{S}_u \tilde{\Xi} |\nabla \tilde{b}| + \dot{\omega}_{ign}$$

Turbulent flame propagation

Deposition model

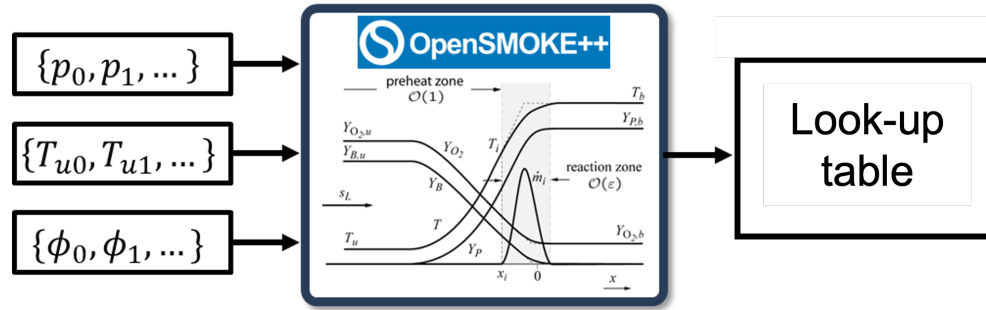
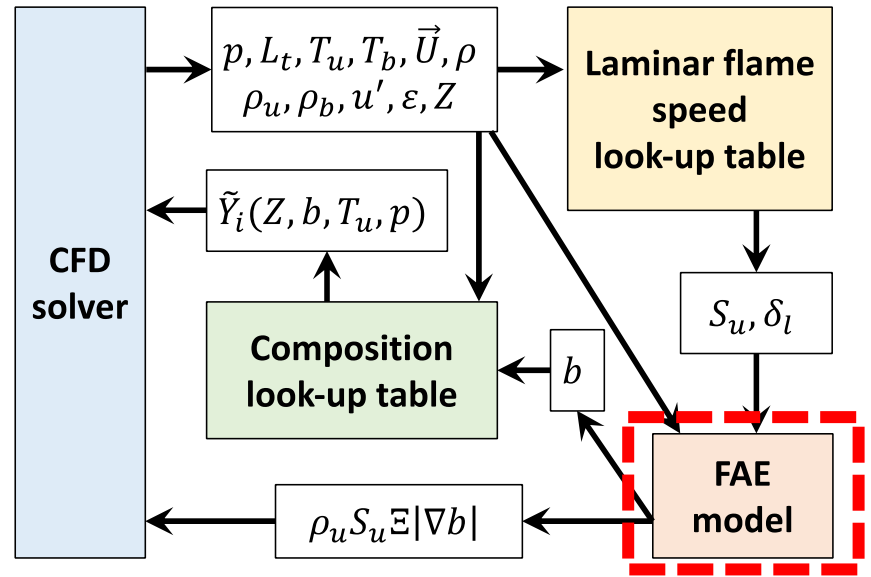
$$\Xi = \frac{S_t}{S_u} = f(\Xi_{eq}^*) \quad \Xi_{eq}^* = 1 - \frac{a_4 b_3^2 L_t}{2 b_1 \delta_l} + \left[\left(\frac{a_4 b_3^2 L_t}{2 b_1 \delta_l} \right)^2 + a_4 b_3^2 \frac{u' L_t}{S_u \delta_l} \right]^{1/2}$$

$$Y_i = b Y_{u,i} + (1 - b) Y_{b,i}$$

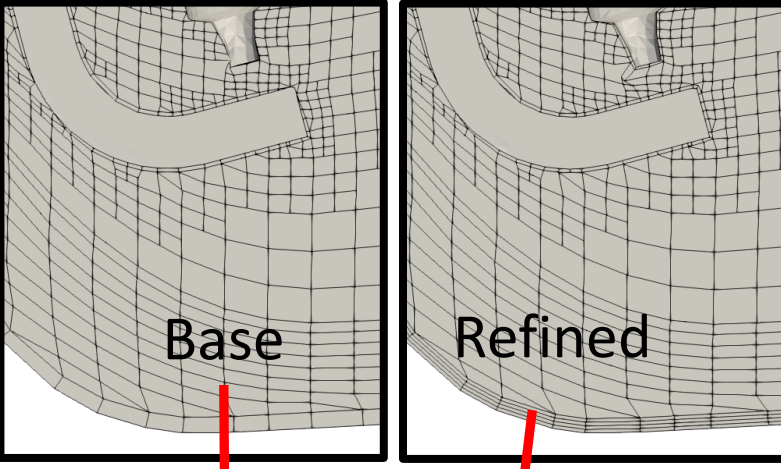
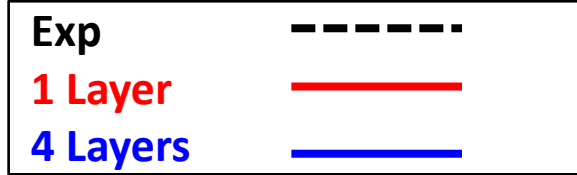
- **Tabulated kinetics** generated with Kéromnès mechanism

- **Laminar flame speed S_u**

$$S_u = C_0 S_{u0} \rightarrow C_0 = 1 + \left(\frac{S_u}{S_{u0}} \Big|_r - 1 \right) \left(\frac{\phi}{\phi_r} \right)^{\gamma_\phi} \left(\frac{T_u}{T_{u,r}} \right)^{\gamma_{T_u}} \left(\frac{p}{p_r} \right)^{\gamma_p}$$



$\lambda = 2.4$



1 Layer

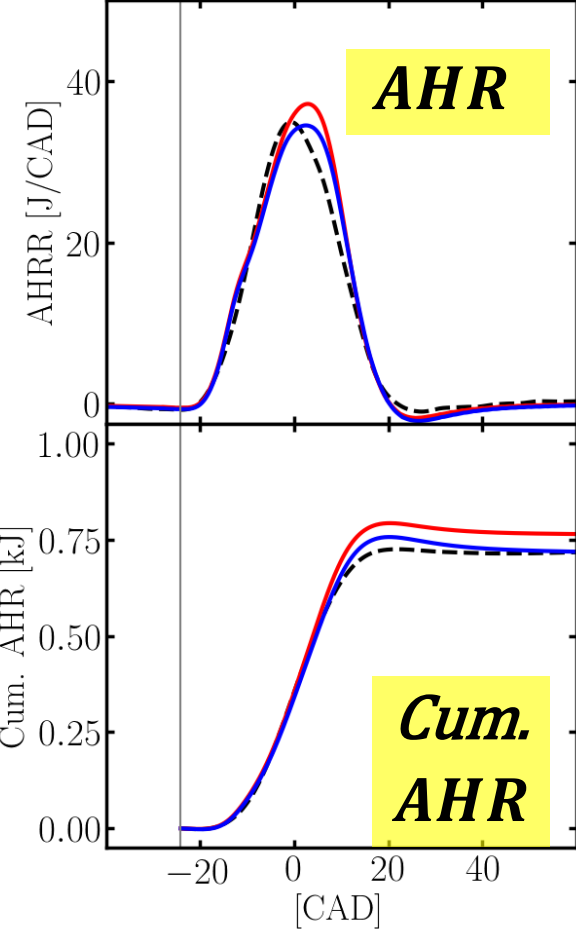
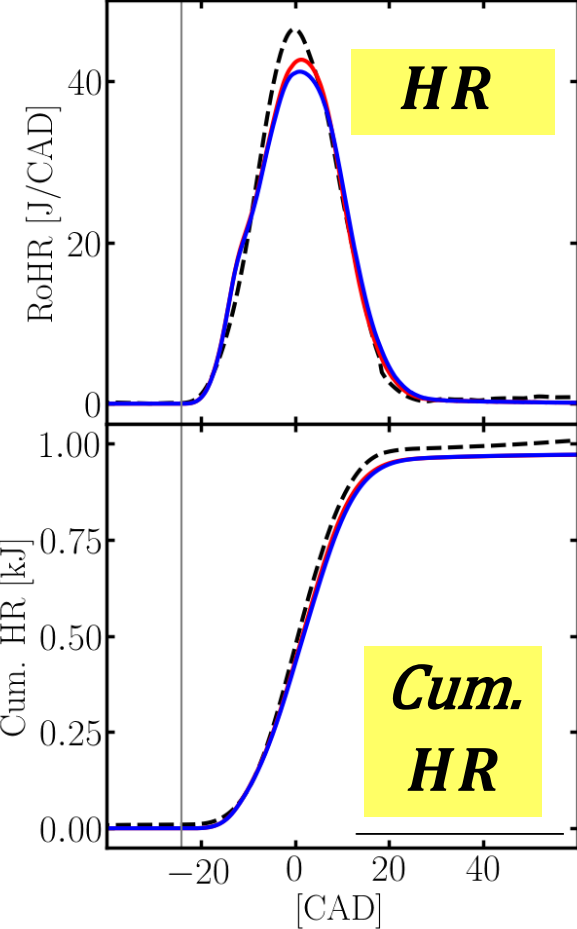
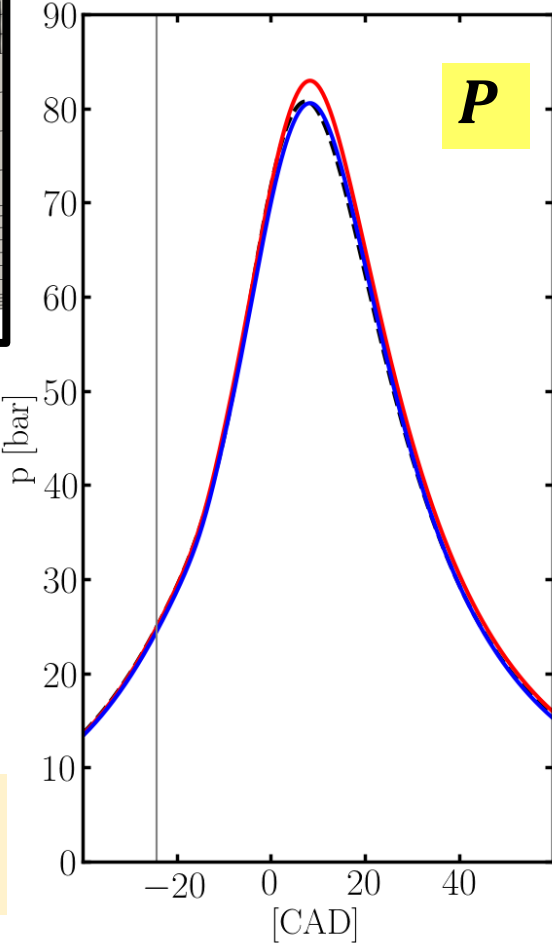
4 Layers

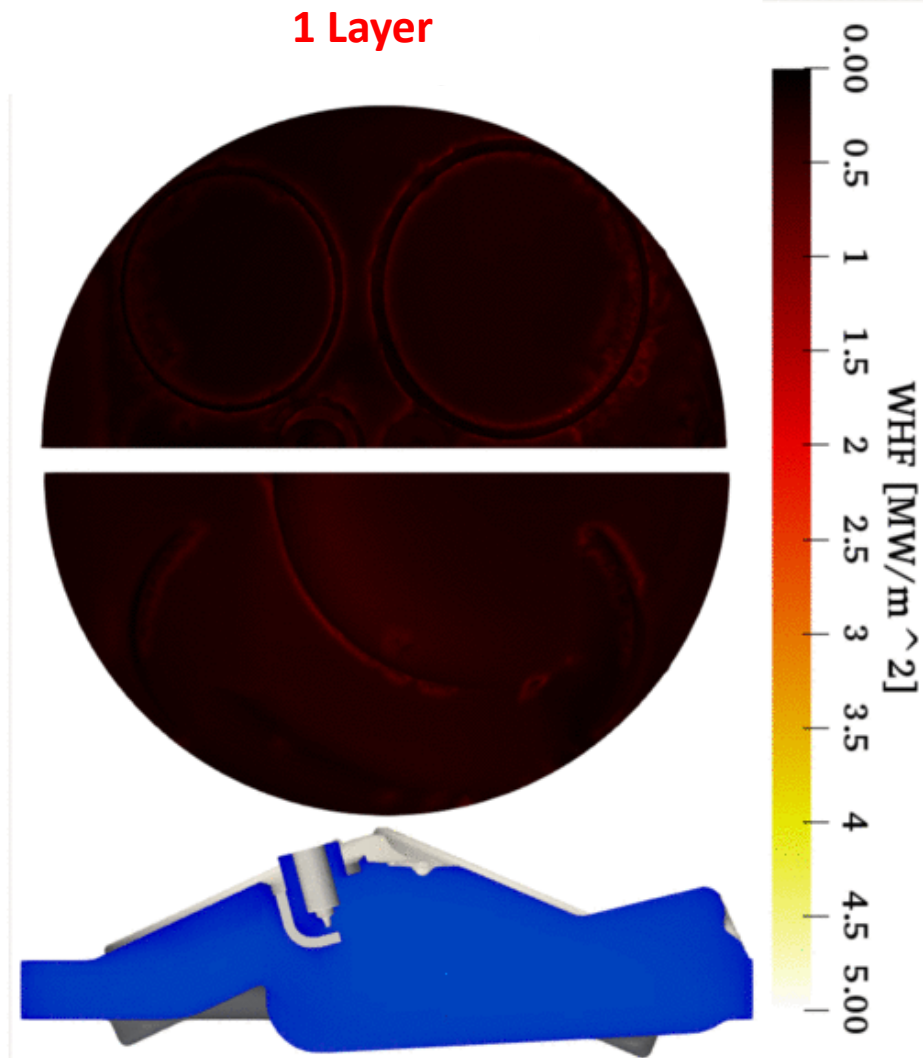
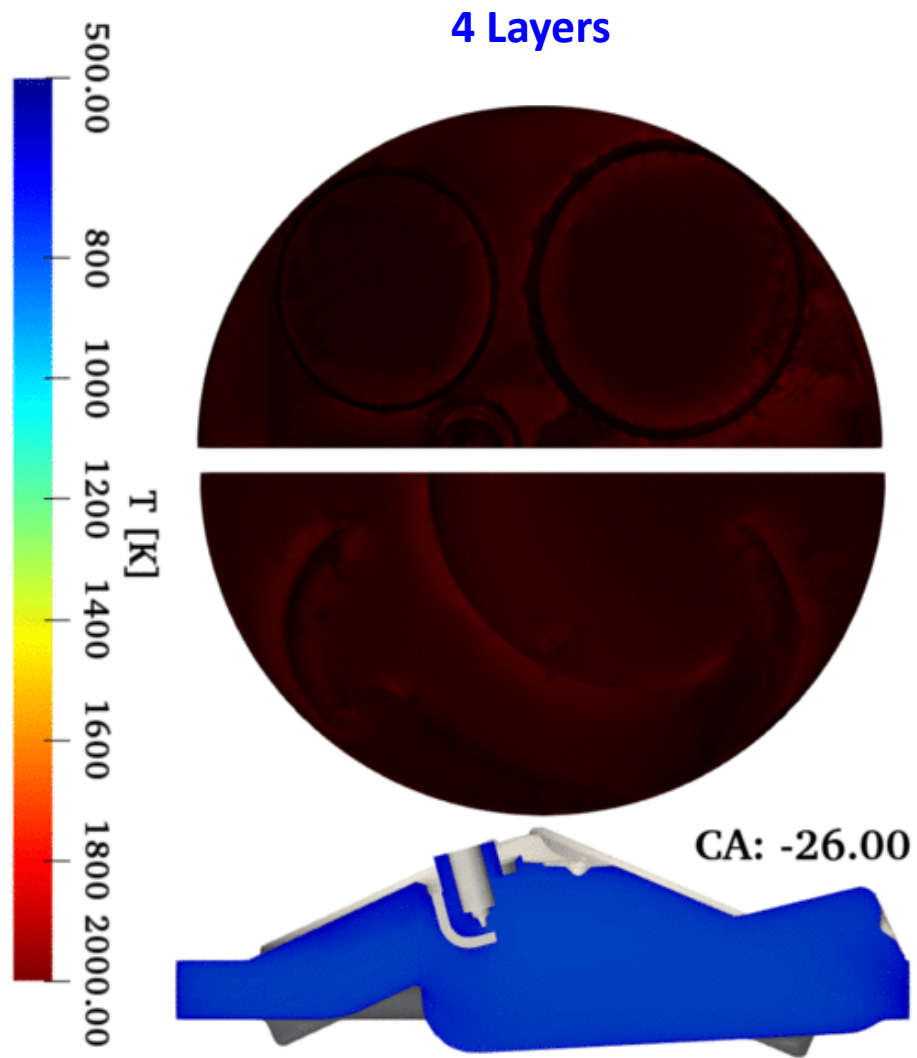
• ↑ n. layers:

■ ↑ wall heat fluxes

↓ AHR peak value

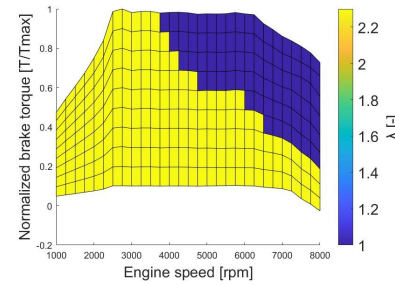
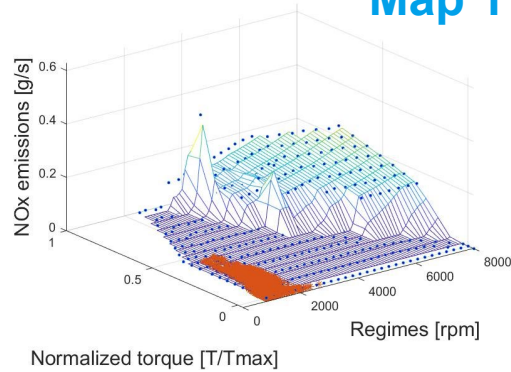
↓ RoHR peak due to flame quenching at walls



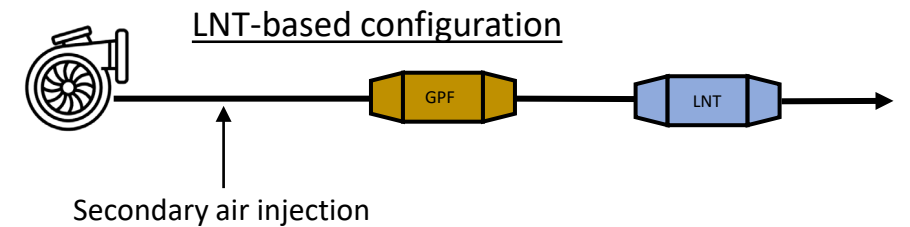
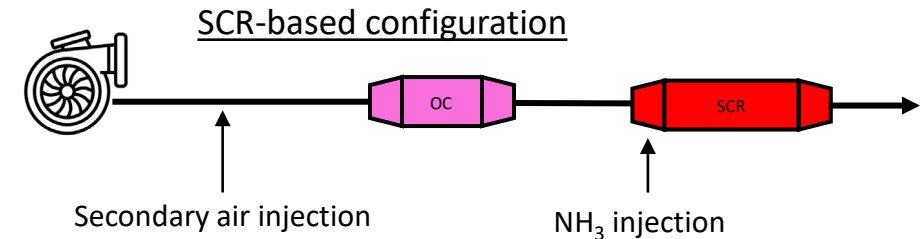
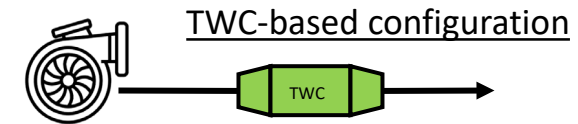
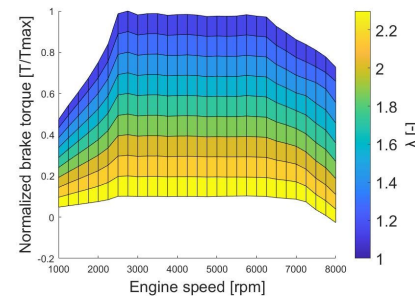
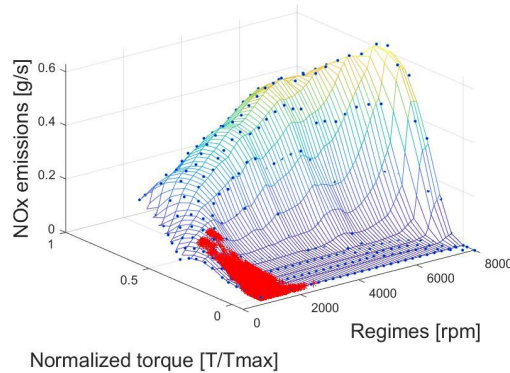


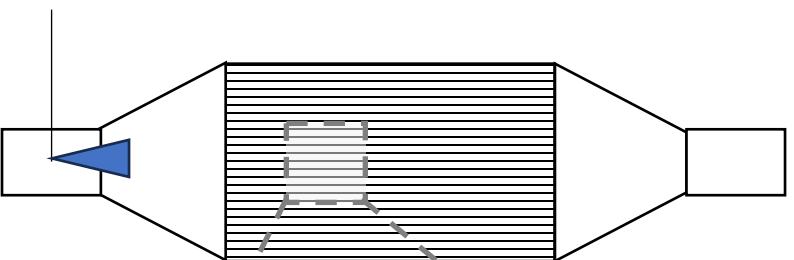
ATS for H2 high-performance engine: coupling engine strategies and ATS design

Map 1

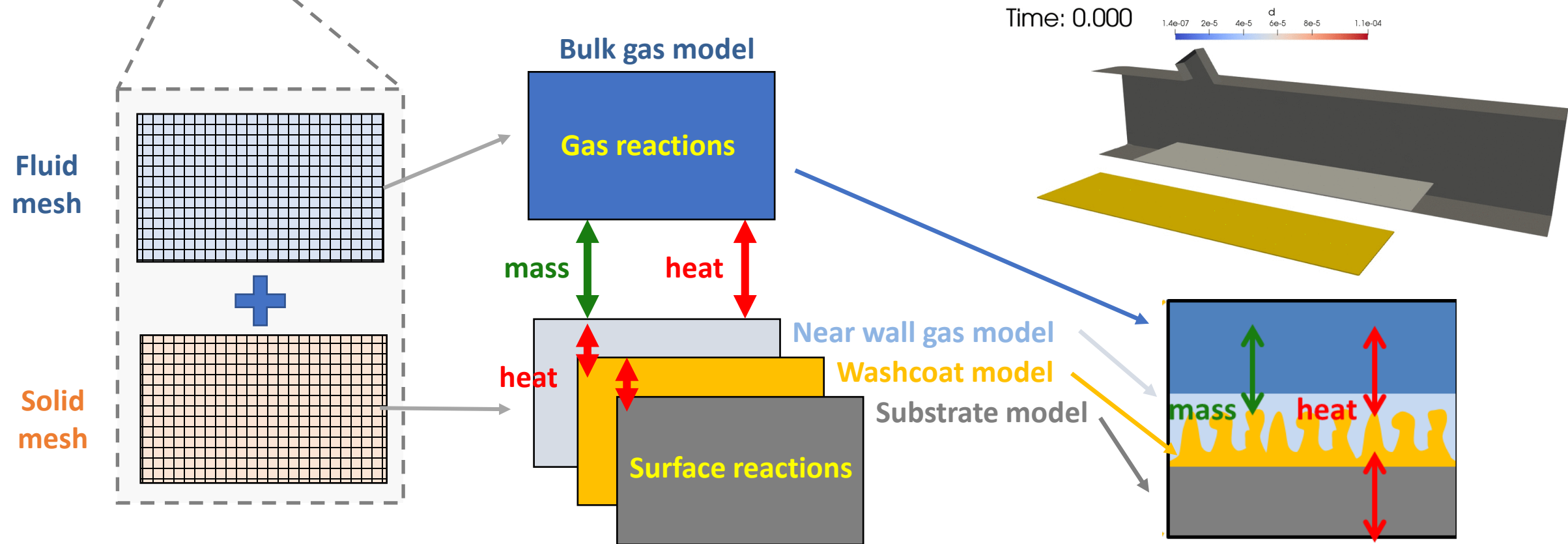


Map 2

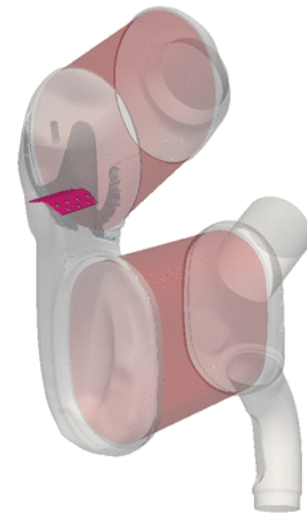
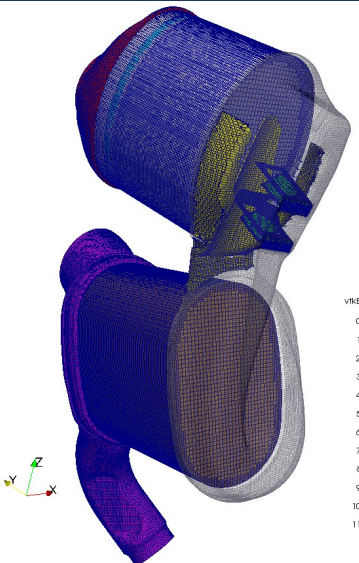
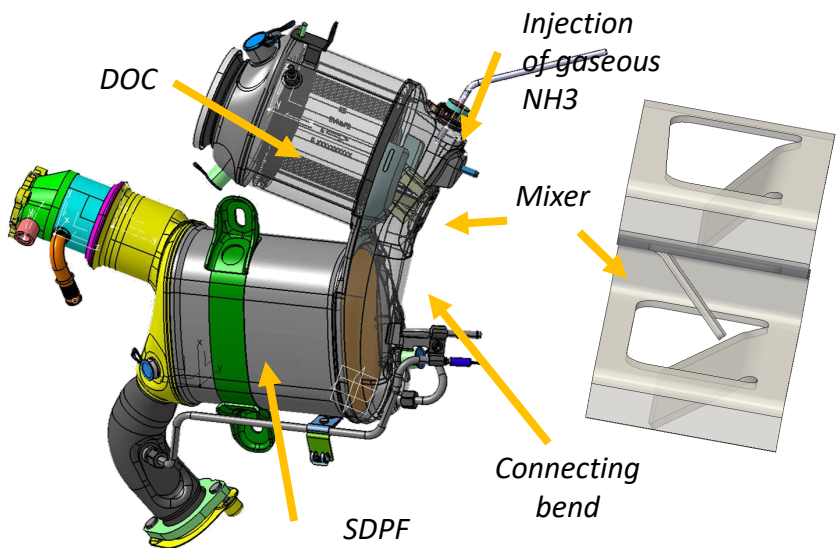




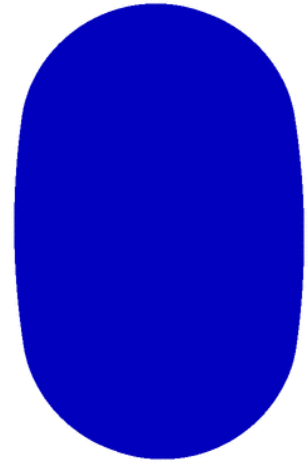
- Overlapping fluid and solid FV meshes
- Solid mesh support the modelling of different zones
- Coupling with lagrangian and eulerian models for tracking of droplets and liquid film



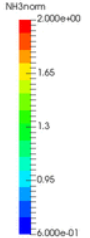
GASEOUS NH3



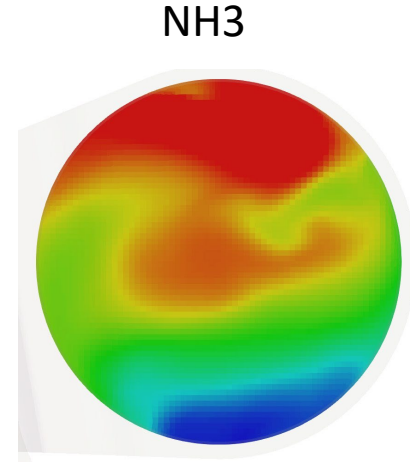
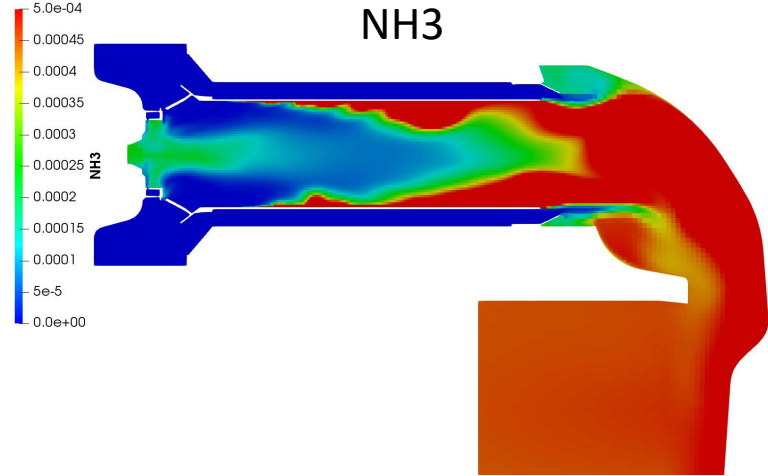
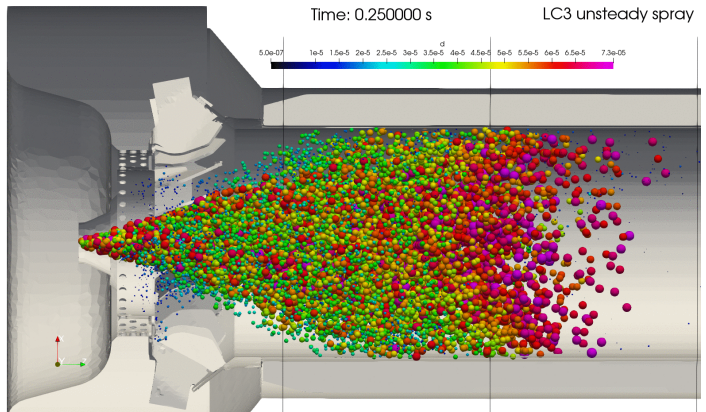
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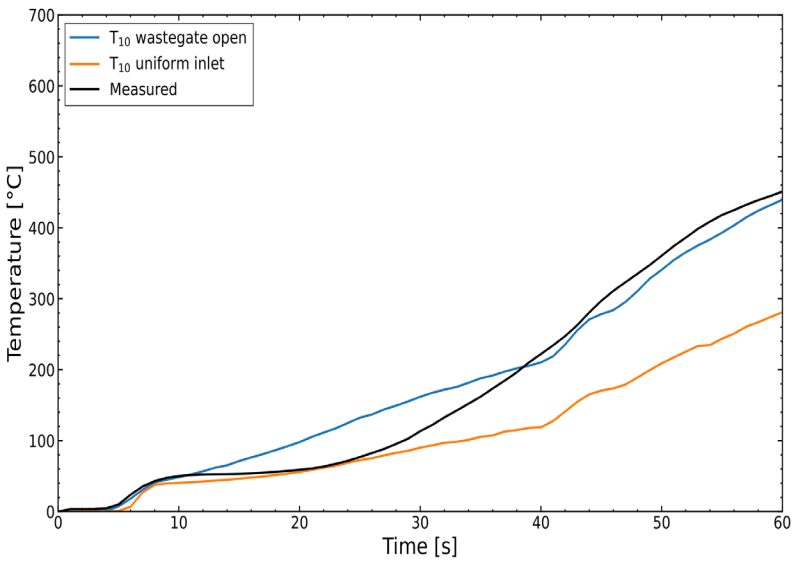
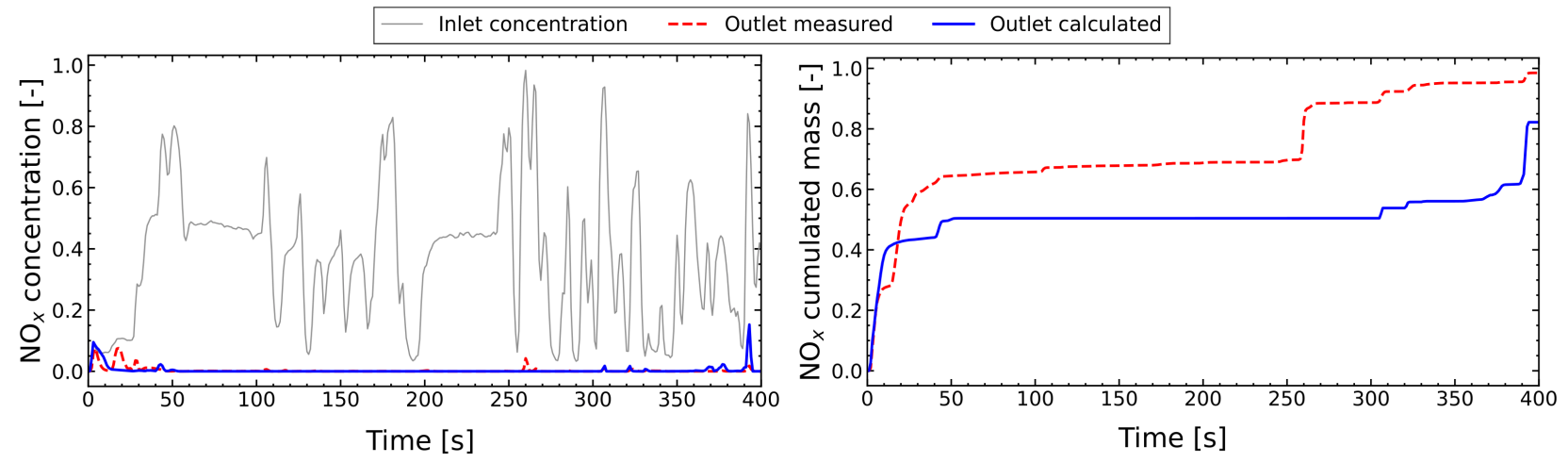
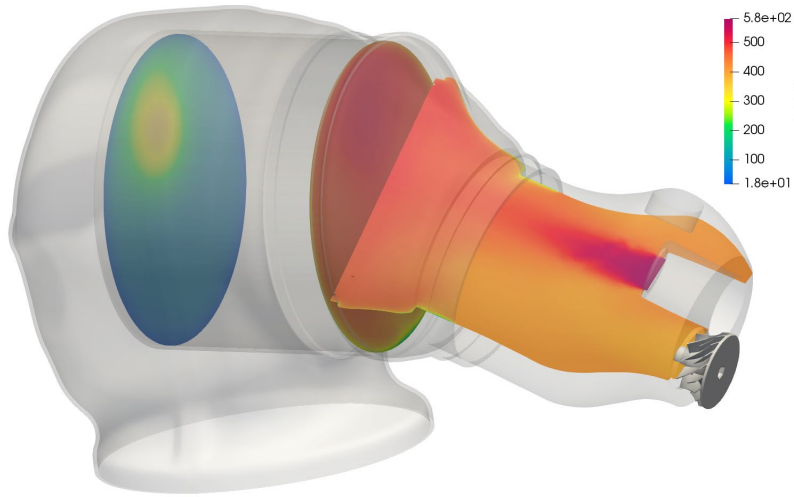


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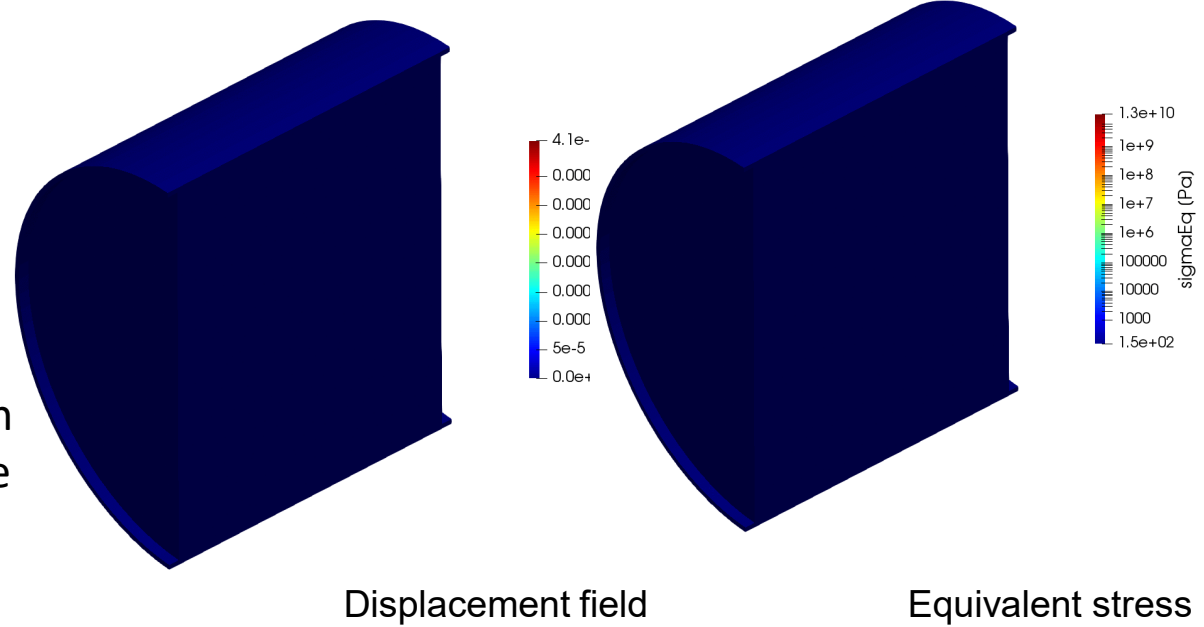


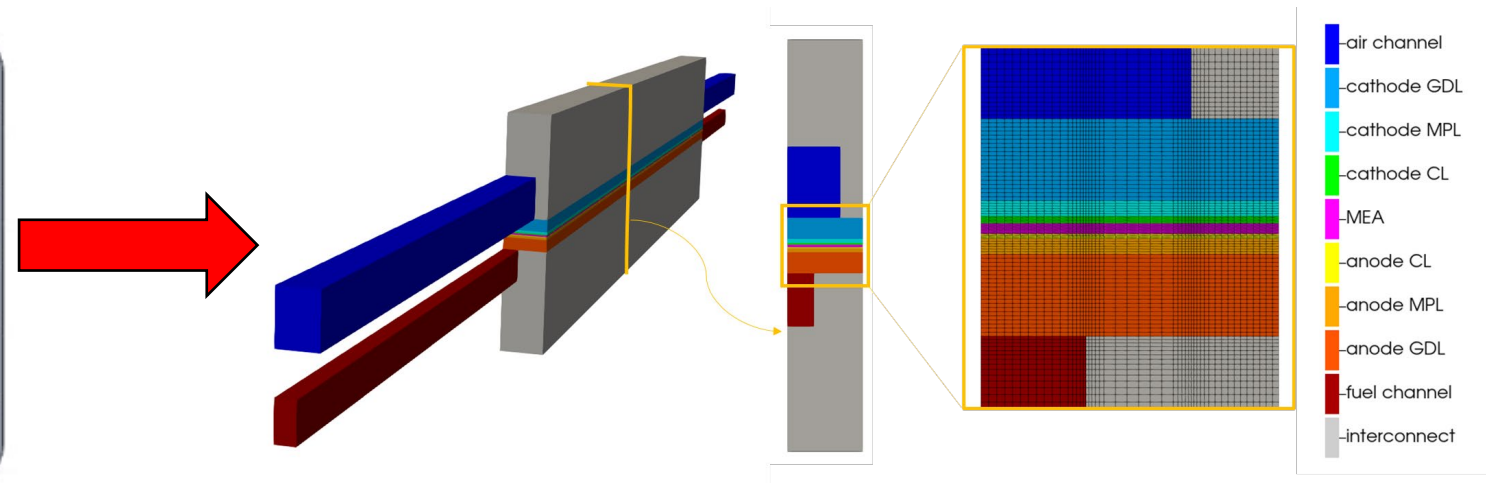
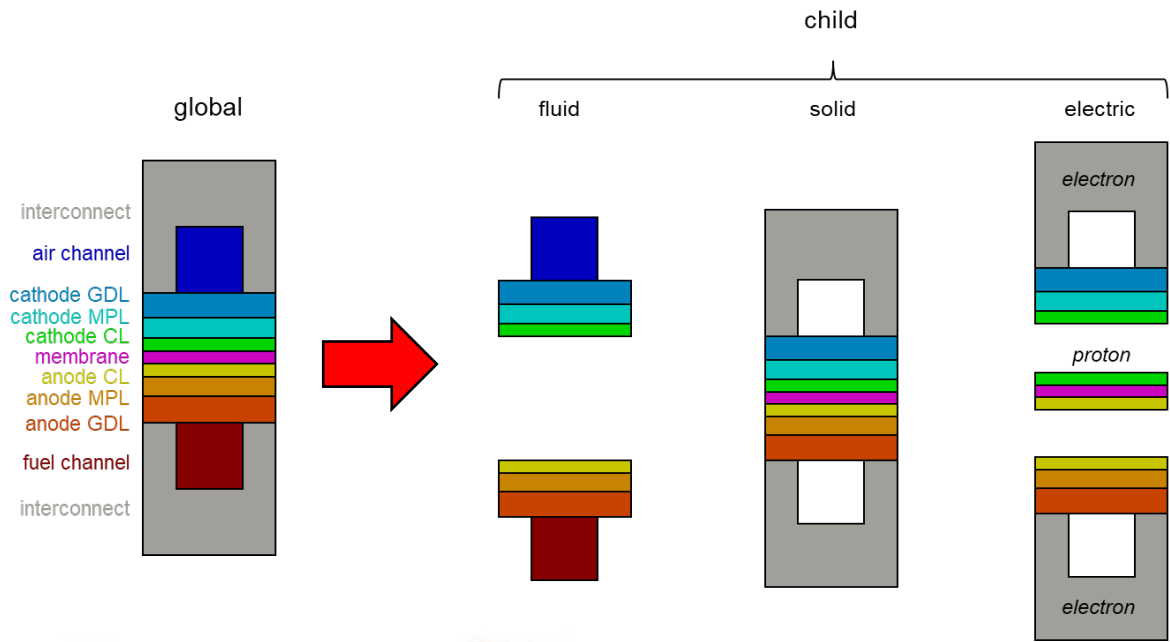
UWS NH3 CARRIER





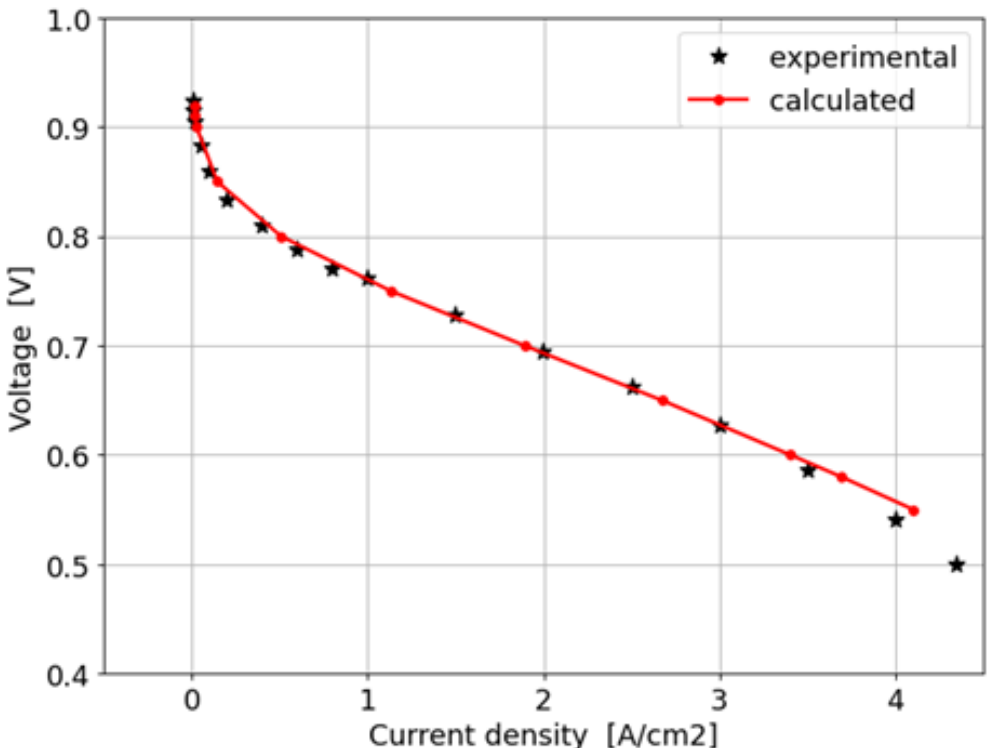
- Water condensation and evaporation submodel
- Stress and deformation caused by temperature gradients





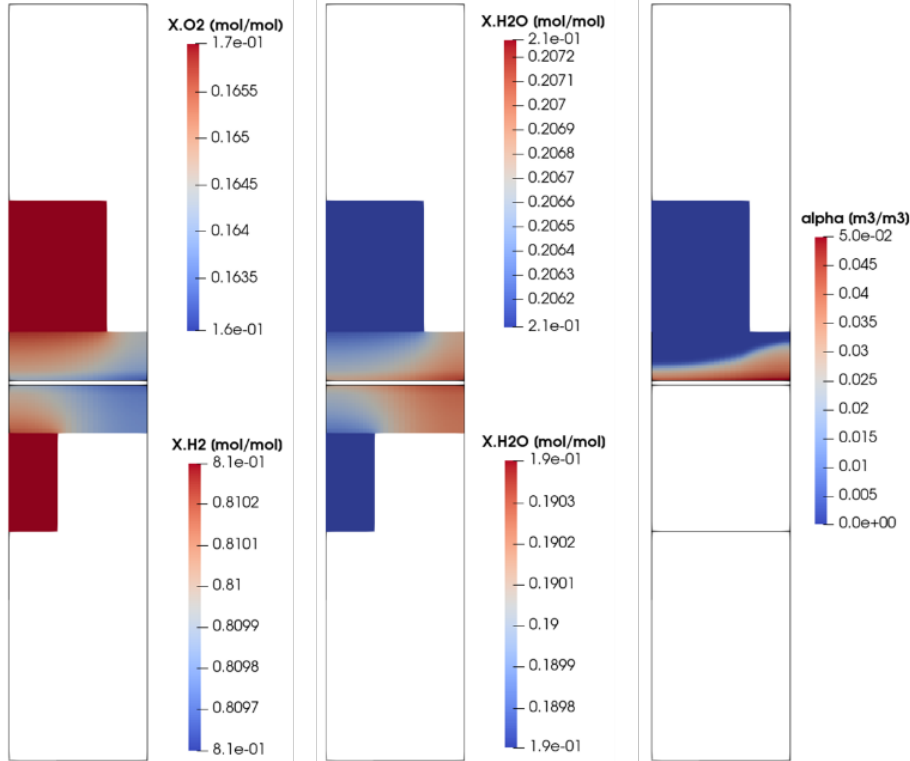
- The model was originally developed by Weber et al. (Juelich) based on the OpenFOAM technology and released under GPL2
- It is based on a multi-region approach for the modeling of fluid, solid and electric region
- Modeling at the channel scale and device scale of PEM fuel cells and electrolyzers
- At PoliMI: implementation of novel models (reaction kinetics, water management) aimed at the study of degradation

Validation: polarization curve



- Calculation of the polarization curve: the implemented PtOx model improve the prediction in the kinetic region (low current density)

Detailed analysis of reactants distribution



- Prediction of reactants distribution, water formation and liquid distribution in the GDL / MPL zones

Aknowledgments

- This presentation has collected the results of the whole research group ICEGroup at PoliMi:

Prof. Angelo Onorati

Prof. Gianluca D'Errico

Prof. Tommaso Lucchini

Prof. Augusto Della Torre

Dr. Lorenzo Sforza

Dr. Giovanni Gianetti

Dr. Andrea Marinoni

Ing. Loris Barillari

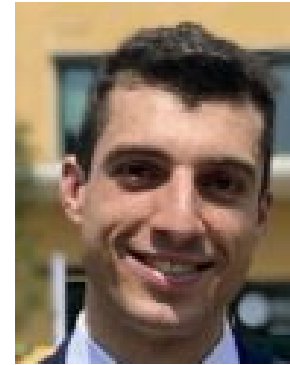
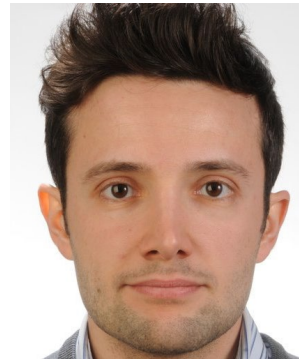
Ing. Federico Ramognino

Ing. Alberto Ballerini

Ing. Alessandro Nodi

Ing. Andrea Sartirana

Ing. Margherita Bulgarini





The sixth edition of the "Two Day Meeting on Propulsion Simulations using OpenFOAM Tecnology" co-organized by the Internal Combustion Engine Group at Politecnico di Milano and the STFS group at TU Darmstadt will be held at Department of Energy of Politecnico di Milano on 11th and 12th March 2024.

This Two-Day Meeting is aimed at engineers and researchers who use OpenFOAM technology to simulate propulsion systems. Similar to previous events, the primary focus is on internal combustion engines, with an emphasis on alternative and sustainable fuels. Furthermore, a new session on fuel cells and BEV has been added. Presentations from universities, research institutes and industries will address the state of the art, real applications and specific developments related to:

- in-cylinder flows: gas exchange, fuel-air mixing, combustion, pollutant formation;
- intake and exhaust systems: after-treatment, unsteady flows, silencers;
- fuel cells and BEV powertrain cooling modeling.





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THANK YOU



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